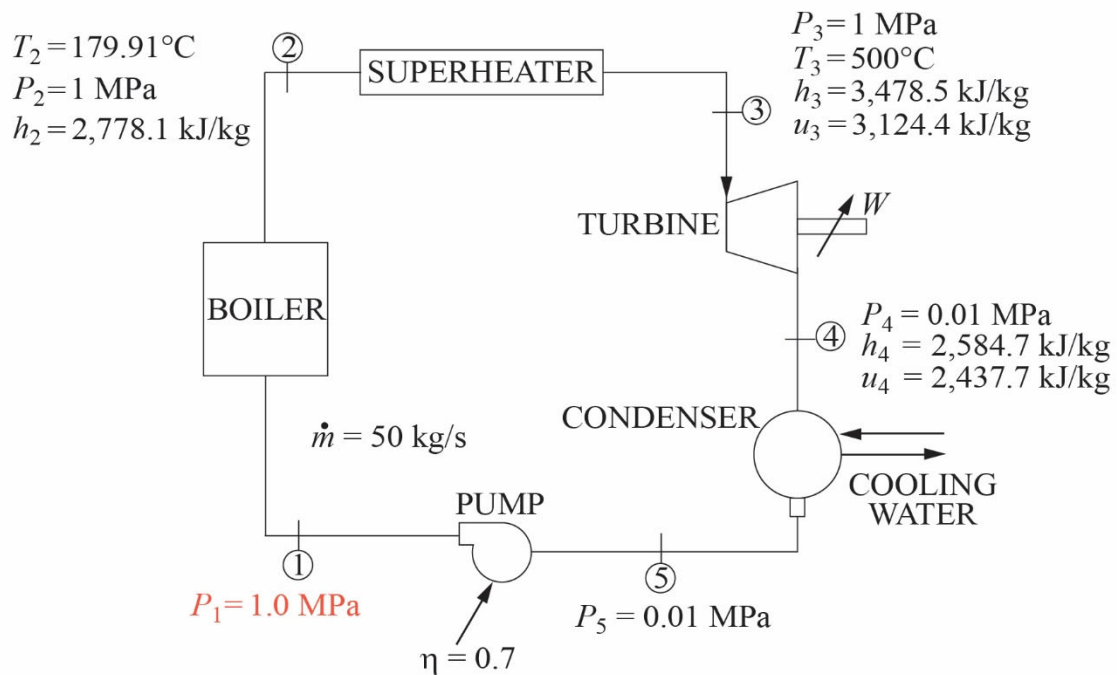


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Revisions are shown in red.

**Question 67, p. 44**

The illustration should be shown as follows:



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**Question 91, p. 59**

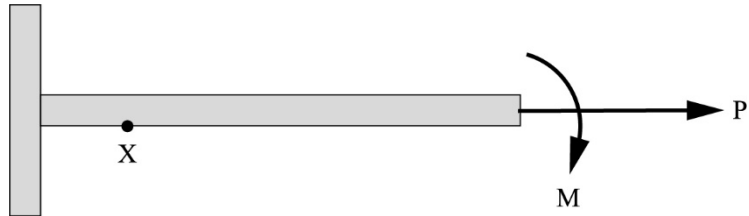
The question should be shown as follows:

Yielding is considered failure for the ductile beam shown. The following data apply:

Yielding first occurs at Point X.

$$S_y = 34 \text{ ksi}$$

$$\sigma_2 = 0 \text{ ksi}$$



The table below shows various calculated values for  $\sigma_1$  and  $\sigma_3$ .

Based on maximum shear stress theory, select the column(s) that show the values at which failures will occur.

Stress	A	B	C	D	E	F
$\sigma_1$ (ksi)	35	24	45	60	18	82
$\sigma_3$ (ksi)	-35	0	-35	-24	-62	-62

**Solution Table, p. 66**

No. 91—A, C, D, E, F

**Solution 12, p. 70**

Refer to the Intellectual Property section in the Ethics chapter of the *FE Reference Handbook*.

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**THE CORRECT ANSWER IS: D**

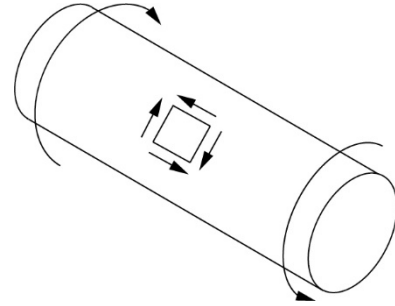
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**Solution 46, p. 91**

Refer to the Torsion section in the Mechanics of Materials chapter of the *FE Reference Handbook*.

The polar moment of inertia for a solid cylinder is

$$J = \frac{\pi r^4}{2} = \frac{\pi \left(\frac{d}{2}\right)^4}{2} = \frac{\pi d^4}{2(2)^4} = \frac{\pi d^4}{2(16)} = \frac{\pi d^4}{32}$$



The equation for the shear stress

$$\tau = \frac{Tr}{J} = \frac{T\left(\frac{d}{2}\right)}{J}$$

Substituting for polar inertia into the equation for the shear stress gives

$$\tau = \frac{Tr}{J} = \frac{T\left(\frac{d}{2}\right)}{\frac{\pi d^4}{32}} = \frac{16T}{\pi d^3}$$

Solving for torque gives

$$T = \frac{\pi d^3 \tau}{16} = \frac{\pi(0.2)^3 (840 \times 10^3)}{16}$$

$$T = 1,319 \text{ N}\cdot\text{m}$$

**THE CORRECT ANSWER IS: C**

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**Solution 91, p. 112**

The solution should read as follows:

Refer to the Static Loading Failure Theories section in the Mechanical Engineering chapter of the *FE Reference Handbook*.

Maximum shear stress theory is used for the ductile material. **Note:**  $\sigma_1 \geq \sigma_2 \geq \sigma_3$

Calculate  $\tau_{\max} = (\sigma_1 - \sigma_3)/2$  and compare with  $S_y/2$ .

Stress	A	B	C	D	E	F
Calculated $\tau_{\max}$	35	12	40	42	40	72
$\tau_{\max} \geq 17$ <b>ksi</b>	fails	12—ok	fails	fails	fails	fails

**THE CORRECT ANSWERS ARE: A, C, D, E, F**

**Solution 95, p. 114**

Refer to the Hooke's Law section in the Mechanics of Materials chapter of the *FE Reference Handbook*.

The formula for the total longitudinal strain without a temperature rise is:

$$\epsilon_{\text{axial}} = \frac{1}{E} (\sigma_l - \nu(\sigma_t + \sigma_r)) = \frac{1}{210 \times 10^3 \text{ MPa}} (23.1 \text{ MPa} - 0.24(46.2 \text{ MPa} + 0)) = 5.72 \times 10^{-5}$$

This must be converted to displacement using the following formula:

$$\epsilon_{\text{axial}} = \frac{\delta l}{l}, \text{ where } l \text{ is the length of the section under consideration}$$

$$\delta l = \epsilon_{\text{axial}} \times l$$

$$= 5.72 \times 10^{-5} \times 1,000 \text{ mm}$$

$$= 0.0572 \text{ mm}$$

**THE CORRECT ANSWER IS: A**

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**Solution 99, p. 117**

The first line of the solution should read as follows:

Use the Failure by Pure Shear equation from the Joining Methods section in the Mechanical Engineering chapter of the *FE Reference Handbook*.